# Reciprocating Pump - Useful equations

## **Discharge**

$$Q = \frac{LAN}{60}$$
 for a single acting pump

$$Q = \frac{2LAN}{60}$$
 for a double acting pump

# Work done by a reciprocating pump per second

$$W = \rho gQH = \frac{\rho gLAN(hs + hd)}{60}$$
 for a single acting pump

$$W = 2\rho gQH = 2\rho gLAN(h"s" + hd)/60$$
 for a double acting pump

## Slip

$$slip = Qthe - Qact$$

$$%slip = \frac{Qthe - Qact}{Qthe} * 100$$

# Pressure head due to acceleration in the suction and delivery pipe

$$has = \frac{l_s}{g} \frac{A}{a_s} r \omega^2 cos\theta$$

$$had = \frac{l_d}{g} \frac{A}{a_d} r \omega^2 cos\theta$$

## Pressure head inside the cylinder due to acceleration

#### **Suction Stroke**

starting of the suction stroke,  $= h_s + h_{as}$  below atmospheric pressure

$$= Hatm - (h_s + h_{as})$$
 absolute pressure

 $middle\ of\ the\ suction\ stroke, =\ h_s\ below\ atmospheric\ pressure$ 

$$= Hatm - h_s$$
 absolute pressure

end of the suction stroke, =  $h_s - h_{as}$  below atmospheric pressure

$$= Hatm - (h_s - h_{as})$$
 absolute pressure

## **Delivery Stroke**

Starting of the delivery stroke,  $= h_d + h_{ad}$  above atmospheric pressure

$$= Hatm + (h_d + h_{ad})$$
 absolute pressure

middle of the delivery stroke,  $= h_d$  above atmospheric pressure

$$= Hatm + h_d$$
 absolute pressure

end of the delivery stroke,  $= h_d - h_{ad}$  above atmospheric pressure

$$= Hatm + (h_d - h_{ad})$$
 absolute pressure

Work done by the pump per second due to acceleration

$$W = \frac{\rho g A L N}{60} (h_s + h_d), \quad \text{single acting pump}$$

Loss of head due to friction in the suction and delivery pipe

$$hf = \frac{4flv^2}{2gd}$$

$$=\frac{4fl}{2gd}\left(\frac{A}{a}r\omega sin\theta\right)^{2}, general\ equation\ (substitute\ l,d\ and\ a\ corresponding\ to\ the\ required\ side)$$

Pressure head inside the cylinder due to acceleration and friction

#### **Suction Stroke**

starting of the suction stroke,  $= h_s + h_{as}$  below atmospheric pressure

$$= Hatm - (h_s + h_{as})$$
 absolute pressure

 $middle\ of\ the\ suction\ stroke, =\ h_s + h_{fs}\ below\ atmospheric\ pressure$ 

$$= Hatm - (h_s + h_{fs})$$
 absolute pressure

end of the suction stroke,  $= h_s - h_{as}$  below atmospheric pressure

$$= Hatm - (h_s - h_{as})$$
 absolute pressure

## **Delivery Stroke**

Starting of the delivery stroke,  $= h_d + h_{ad}$  above atmospheric pressure

$$= Hatm + (h_d + h_{ad})$$
 absolute pressure

middle of the delivery stroke, =  $h_d + h_{fd}$  above atmospheric pressure

$$= Hatm + (h_d + h_{fd})$$
 absolute pressure

end of the delivery stroke,  $= h_d - h_{ad}$  above atmospheric pressure

$$= Hatm + (h_d - h_{ad})$$
 absolute pressure

Work done by the pump per second due to acceleration and friction is given by

$$W = \frac{\rho g A L N}{60} \left( h_s + h_d + \frac{2}{3} h_{fs} + \frac{2}{3} h_{fd} \right), single \ acting \ pump$$

# Maximum Speed of a reciprocating pump

At the suction side,

$$H_{atm}-(h_s+h_{as})=\ h_{sep}$$
 ,  $absolute$  At the delivery side,

$$H_{atm} + (h_d - h_{ad}) = h_{sep}$$
 , absolute

Find the speed corresponding to suction side as well as delivery side.

Take the minimum speed of the two as the maximum speed at which pump can be operated without separation.

## Air Vessel

Mean velocity for a single acting reciprocating pump

$$\bar{V} = \frac{A \, r \omega}{a \, \pi}$$

Work done by reciprocating pump with air vessels fitted to the suction and delivery pipes

$$\approx \frac{\rho g LAN}{60} \left[ h_s + h_d + h_{fs} + h_{fd} \right]$$